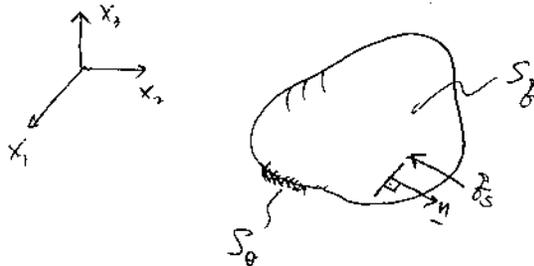


Lecture 16 - F.E. analysis of Navier-Stokes fluids

### Incompressible flow with heat transfer

We recall heat transfer for a solid:

Reading:  
Sec.  
7.1-7.4,  
Table 7.3



#### Governing differential equations

$$(k\theta_{,i})_{,i} + q^B = 0 \quad \text{in } V \tag{16.1}$$

$$\theta|_{S_\theta} \text{ is prescribed, } k \frac{\partial \theta}{\partial n}|_{S_q} = q^S|_{S_q} \tag{16.2}$$

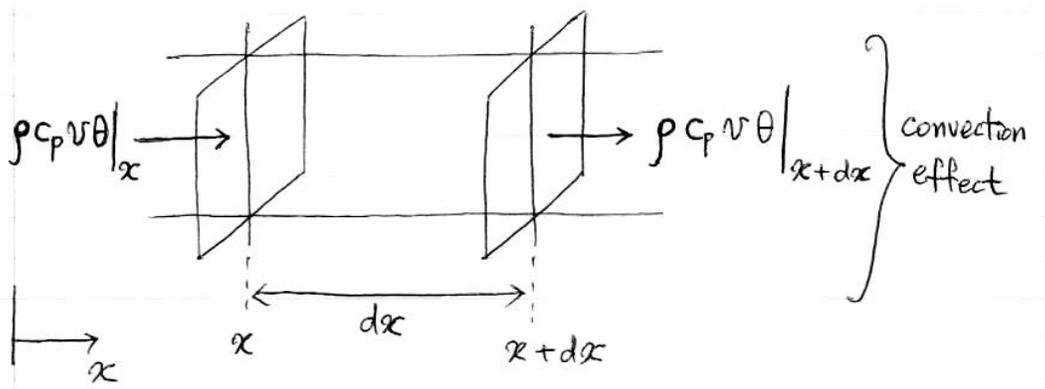
$$S_\theta \cup S_q = S \quad S_\theta \cap S_q = \emptyset \tag{16.3}$$

#### Principle of virtual temperatures

$$\int_V \bar{\theta}_{,i} k \theta_{,i} dV = \int_V \bar{\theta} q^B dV + \int_{S_q} \bar{\theta}^S q^S dS_q \tag{16.4}$$

for arbitrary continuous  $\bar{\theta}(x_1, x_2, x_3)$  zero on  $S_\theta$

For a fluid, we use the Eulerian formulation.



$$\rho c_p v \theta|_x - \left\{ \rho c_p v \theta|_x + \frac{\partial}{\partial x} (\rho c_p v \theta) dx \right\} + \text{conduction} + \text{etc} \quad (16.5)$$

In general 3D, we have an additional term for the left hand side of (16.1):

$$-\nabla \cdot (\rho c_p \mathbf{v} \theta) = -\rho c_p \nabla \cdot (\mathbf{v} \theta) = -\rho c_p (\nabla \cdot \mathbf{v}) \theta - \underbrace{\rho c_p (\mathbf{v} \cdot \nabla) \theta}_{\text{term (A)}} \quad (16.6)$$

where  $\nabla \cdot \mathbf{v} = 0$  in the incompressible case.

$$\nabla \cdot \mathbf{v} = v_{i,i} = \text{div}(\mathbf{v}) = 0 \quad (16.7)$$

So (16.1) becomes

$$(k\theta_{,i})_{,i} + q^B = \rho c_p \theta_{,i} v_i \Rightarrow (k\theta_{,i})_{,i} + (q^B - \rho c_p \theta_{,i} v_i) = 0 \quad (16.8)$$

Principle of virtual temperatures is now (use (16.4))

$$\int_V \bar{\theta}_{,i} k \theta_{,i} dV + \int_V \bar{\theta} (\rho c_p \theta_{,i} v_i) dV = \int_V \bar{\theta} q^B dV + \int_{S_q} \bar{\theta} q^S dS_q \quad (16.9)$$

### Navier-Stokes equations

- *Differential form*

$$\tau_{ij,j} + f_i^B = \rho v_{i,j} v_j \quad (16.10)$$

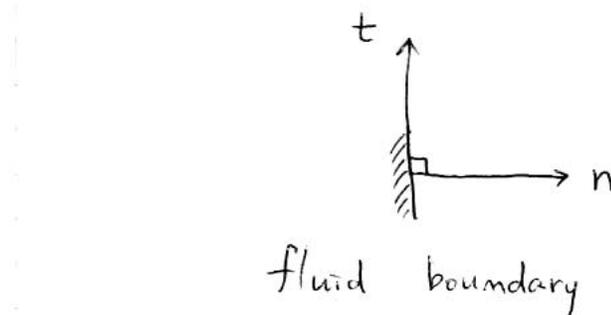
with  $\rho v_{i,j} v_j$  like term (A) in (16.6) =  $\rho(\mathbf{v} \cdot \nabla) \mathbf{v}$  in  $V$ .

$$\tau_{ij} = -p\delta_{ij} + 2\mu e_{ij} \quad e_{ij} = \frac{1}{2} \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \quad (16.11)$$

- *Boundary conditions* (need be modified for various flow conditions)

$$\tau_{ij} n_j = f_i^{S_f} \text{ on } S_f \quad (16.12)$$

Mostly used as  $f_n = \tau_{nn} =$  prescribed,  $f_t =$  unknown with possibly  $\frac{\partial v_n}{\partial n} = \frac{\partial v_t}{\partial n} = 0$  (outflow or inflow conditions).



And  $v_i$  prescribed on  $S_v$ , and  $S_v \cup S_f = S$  and  $S_v \cap S_f = \emptyset$ .

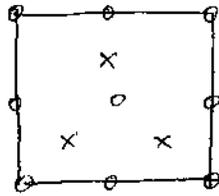
- Variational form

$$\int_V \bar{v}_i \rho v_{i,j} v_j dV + \int_V \bar{e}_{ij} \tau_{ij} dV = \int_V \bar{v}_i f_i^B dV + \int_{S_f} \bar{v}_i^{S_f} f_i^{S_f} dS_f \tag{16.13}$$

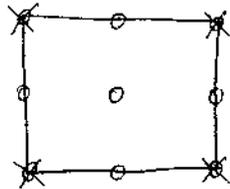
$$\int_V \bar{p} \nabla \cdot \mathbf{v} dV = 0 \tag{16.14}$$

- F.E. solution

We interpolate  $(x_1, x_2, x_3), v_i, \bar{v}_i, \theta, \bar{\theta}, p, \bar{p}$ . Good elements are



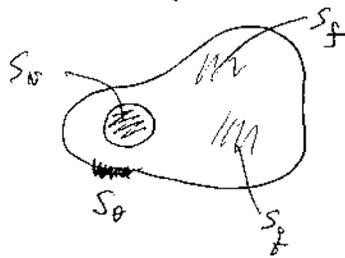
x: linear pressure  
o: biquadratic velocities  
 $(Q_2, P_1), 9/3$  element



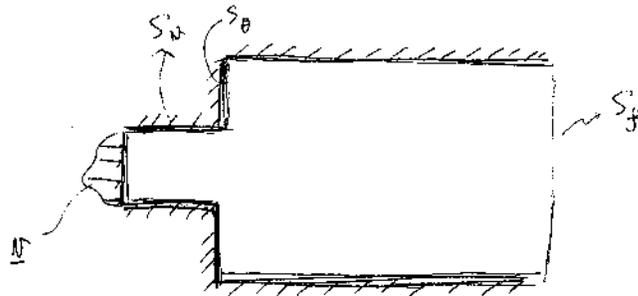
9/4c element

Both satisfy the inf-sup condition.

So in general,



Example:

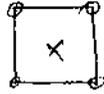


For  $S_f$  e.g.

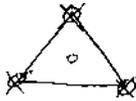
$$\tau_{nn} = 0, \quad \frac{\partial v_t}{\partial n} = 0; \tag{16.15}$$

and  $\frac{\partial v_n}{\partial t}$  is solved for. Actually, we frequently just set  $p = 0$ .

Frequently used is the 4-node element with constant pressure



It does not strictly satisfy the inf-sup condition. Or use



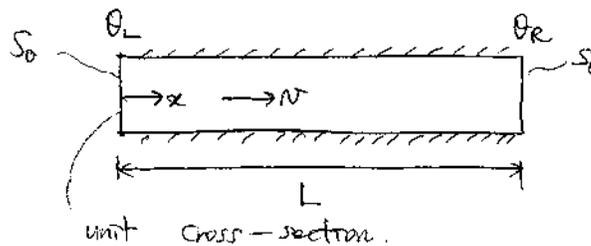
3-node element with a bubble node.  
Satisfies inf-sup condition

Reading:  
Sec. 7.4

1D case of heat transfer with fluid flow,  $v = \text{constant}$

Reading:  
Sec. 7.4.3

$$\text{Re} = \frac{vL}{\nu} \quad \text{Pe} = \frac{vL}{\alpha} \quad \alpha = \frac{k}{\rho c_p} \quad (16.16)$$



$$S_f = \phi$$

- *Differential equations*

$$k\theta'' = \rho c_p \theta' v \quad (16.17)$$

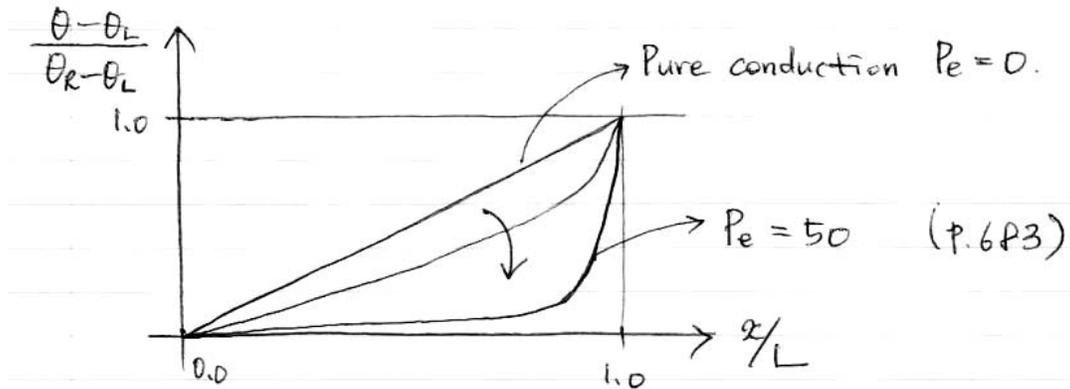
$$\theta|_{x=0} = \theta_L \quad \theta|_{x=L} = \theta_R \quad (16.18)$$

In non-dimensional form

Reading:  
p. 683

$$\boxed{\frac{1}{\text{Pe}} \theta'' = \theta'} \quad (\text{now } \theta'' \text{ and } \theta' \text{ are non-dimensional}) \quad (16.19)$$

$$\Rightarrow \frac{\theta - \theta_L}{\theta_R - \theta_L} = \frac{\exp\left(\frac{\text{Pe}}{L}x\right) - 1}{\exp(\text{Pe}) - 1} \quad (16.20)$$

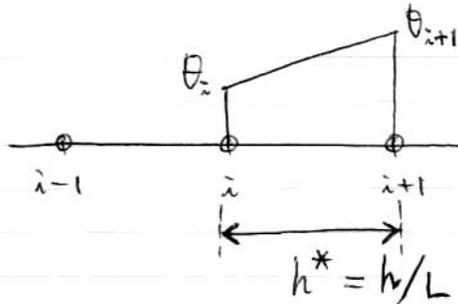


- F.E. discretization

$$\theta'' = \text{Pe}\theta' \quad (16.21)$$

$$\int_0^1 \bar{\theta}' \theta' dx + \text{Pe} \int_0^1 \bar{\theta} \theta' dx = 0 + \{ \text{effect of boundary conditions} = 0 \text{ here} \} \quad (16.22)$$

Using 2-node elements gives



$$\frac{1}{(h^*)^2} (\theta_{i+1} - 2\theta_i + \theta_{i-1}) = \frac{\text{Pe}}{2h^*} (\theta_{i+1} - \theta_{i-1}) \quad (16.23)$$

$$\text{Pe} = \frac{vL}{\alpha} \quad (16.24)$$

Define

$$\text{Pe}^e = \text{Pe} \cdot \frac{h}{L} = \frac{vh}{\alpha} \quad (16.25)$$

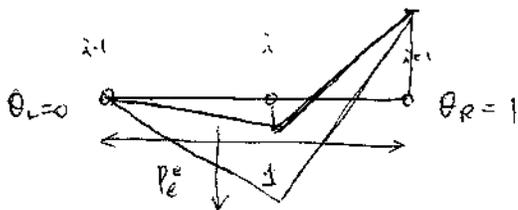
$$\left(-1 - \frac{\text{Pe}^e}{2}\right) \theta_{i-1} + 2\theta_i + \left(\frac{\text{Pe}^e}{2} - 1\right) \theta_{i+1} = 0 \quad (16.26)$$

what is happening when  $\text{Pe}^e$  is large? Assume two 2-node elements only.

$$\theta_{i-1} = 0 \quad (16.27)$$

$$\theta_{i+1} = 1 \quad (16.28)$$

$$\theta_i = \frac{1}{2} \left(1 - \frac{\text{Pe}^e}{2}\right) \quad (16.29)$$



$$\theta_i = \frac{1}{2} \left( 1 - \frac{\text{Pe}^e}{2} \right) \quad (16.30)$$

For  $\text{Pe}^e > 2$ , we have *negative*  $\theta_i$  (unreasonable).

MIT OpenCourseWare  
<http://ocw.mit.edu>

2.094 Finite Element Analysis of Solids and Fluids II  
Spring 2011

For information about citing these materials or our Terms of Use, visit: <http://ocw.mit.edu/terms>.