

Lecture 18 - Solution of F.E. equations

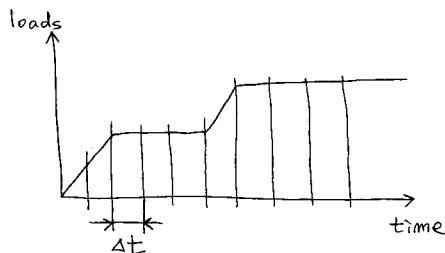
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In structures,

Reading:
Sec. 8.4

$$\mathbf{F}(\mathbf{u}, \mathbf{p}) = \mathbf{R}$$
(18.1)



In heat transfer,

$$\mathbf{F}(\theta) = \mathbf{Q}$$
(18.2)

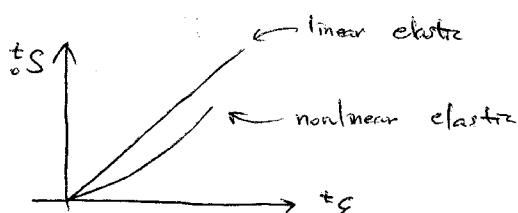
In fluid flow,

$$\mathbf{F}(\mathbf{v}, \mathbf{p}, \theta) = \mathbf{R}$$
(18.3)

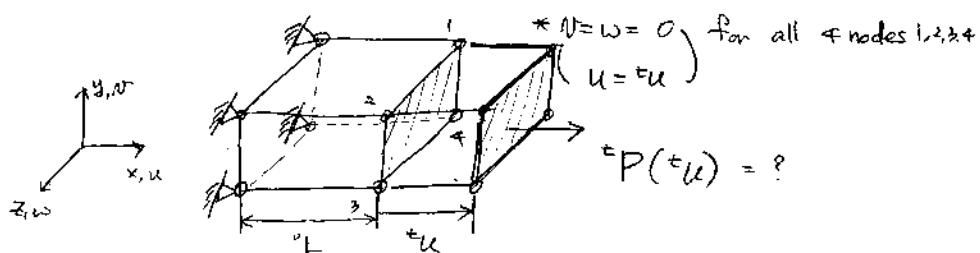
In structures/solids

$$\mathbf{F} = \sum_m \mathbf{F}^{(m)} = \sum_m \int_{0V^{(m)}} {}^t \mathbf{B}_L^{(m)T} {}^t \hat{\mathbf{S}}^{(m)} d^0 V^{(m)}$$
(18.4)

Elastic materials



Example p. 590 textbook



Material law

$${}^t S_{11} = \tilde{E} {}^t \epsilon_{11} \quad (18.5)$$

In isotropic elasticity:

$$\tilde{E} = \frac{E(1-\nu)}{(1+\nu)(1-2\nu)}, \quad (\nu = 0.3) \quad (18.6)$$

$${}^t \epsilon = \frac{1}{2} \left[({}^t U)^2 - I \right] \Rightarrow {}^t \epsilon_{11} = \frac{1}{2} \left[\left(\frac{{}^0 L + {}^t u}{{}^0 L} \right)^2 - 1 \right] = \frac{1}{2} \left[\left(1 + \frac{{}^t u}{{}^0 L} \right)^2 - 1 \right] \quad (18.7)$$

where ${}^t U$ is the stretch tensor.

$${}^t S_{11} = \frac{{}^0 \rho}{{}^t \rho} {}^0 X_{11} {}^t \tau_{11} {}^0 X_{11}^T \quad (18.8)$$

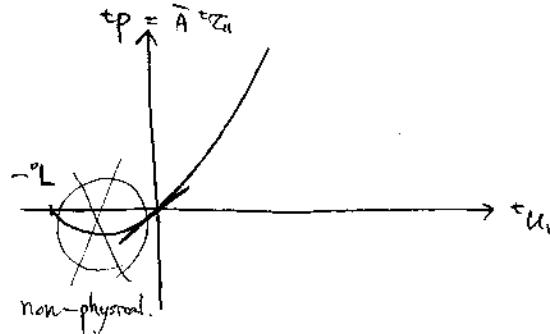
with

$${}^0 X_{11} = \frac{{}^0 L}{{}^0 L + {}^t u}, \quad {}^0 \rho {}^0 L = {}^t \rho {}^t L \quad (18.9)$$

$$\Rightarrow {}^t S_{11} = \frac{{}^t L}{{}^0 L} \left(\frac{{}^0 L}{{}^t L} \right)^2 {}^t \tau_{11} = \frac{{}^0 L}{{}^t L} {}^t \tau_{11} \quad (18.10)$$

$$\therefore \frac{{}^0 L}{{}^t L} {}^t \tau_{11} = \tilde{E} \cdot \frac{1}{2} \left[\left(1 + \frac{{}^t u}{{}^0 L} \right)^2 - 1 \right] \quad (18.11)$$

$$\Rightarrow {}^t \tau_{11} \bar{A} = \boxed{{}^t P = \frac{\tilde{E} \bar{A}}{2} \left[\left(1 + \frac{{}^t u}{{}^0 L} \right)^2 - 1 \right] \left(1 + \frac{{}^t u}{{}^0 L} \right)} \quad (18.12)$$



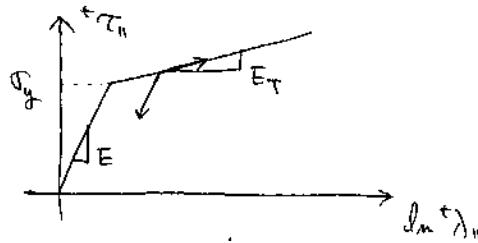
This is because of the material-law assumption (18.5) (okay for small strains ...)

Hyperelasticity

$${}^t W = f(\text{Green-Lagrange strains, material constants}) \quad (18.13)$$

$${}^t S_{ij} = \frac{1}{2} \left(\frac{\partial {}^t W}{\partial {}^t \epsilon_{ij}} + \frac{\partial {}^t W}{\partial {}^t \epsilon_{ji}} \right) \quad (18.14)$$

$${}_0 C_{ijrs} = \frac{1}{2} \left(\frac{\partial {}^t S_{ij}}{\partial {}^t \epsilon_{rs}} + \frac{\partial {}^t S_{ij}}{\partial {}^t \epsilon_{sr}} \right) \quad (18.15)$$

Plasticity

- yield criterion
- flow rule
- hardening rule

$${}^t\boldsymbol{\tau} = {}^{t-\Delta t}\boldsymbol{\tau} + \int_{t-\Delta t}^t d\boldsymbol{\tau} \quad (18.16)$$

Solution of (18.1) (similarly (18.2) and (18.3))

Newton-Raphson Find \mathbf{U}^* as the zero of $f(\mathbf{U}^*)$

$$\mathbf{f}(\mathbf{U}^*) = {}^{t+\Delta t}\mathbf{R} - {}^{t+\Delta t}\mathbf{F} \quad (18.17)$$

$$= \mathbf{f}\left({}^{t+\Delta t}\mathbf{U}^{(i-1)}\right) + \frac{\partial \mathbf{f}}{\partial \mathbf{U}} \Big|_{{}^{t+\Delta t}\mathbf{U}^{(i-1)}} \cdot \left(\mathbf{U}^* - {}^{t+\Delta t}\mathbf{U}^{(i-1)}\right) + H.O.T. \quad (18.18)$$

where ${}^{t+\Delta t}\mathbf{U}^{(i-1)}$ is the value we just calculated and an approximation to \mathbf{U}^* .

Assume ${}^{t+\Delta t}\mathbf{R}$ is independent of the displacements.

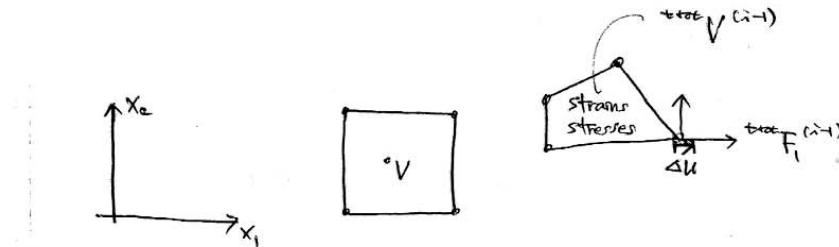
$$\mathbf{0} = \left({}^{t+\Delta t}\mathbf{R} - {}^{t+\Delta t}\mathbf{F}^{(i-1)}\right) - \frac{\partial {}^{t+\Delta t}\mathbf{F}}{\partial \mathbf{U}} \Big|_{{}^{t+\Delta t}\mathbf{U}^{(i-1)}} \cdot \Delta \mathbf{U}^{(i)} \quad (18.19)$$

We obtain

$${}^{t+\Delta t}\mathbf{K}^{(i-1)} \Delta \mathbf{U}^{(i)} = {}^{t+\Delta t}\mathbf{R} - {}^{t+\Delta t}\mathbf{F}^{(i-1)} \quad (18.20)$$

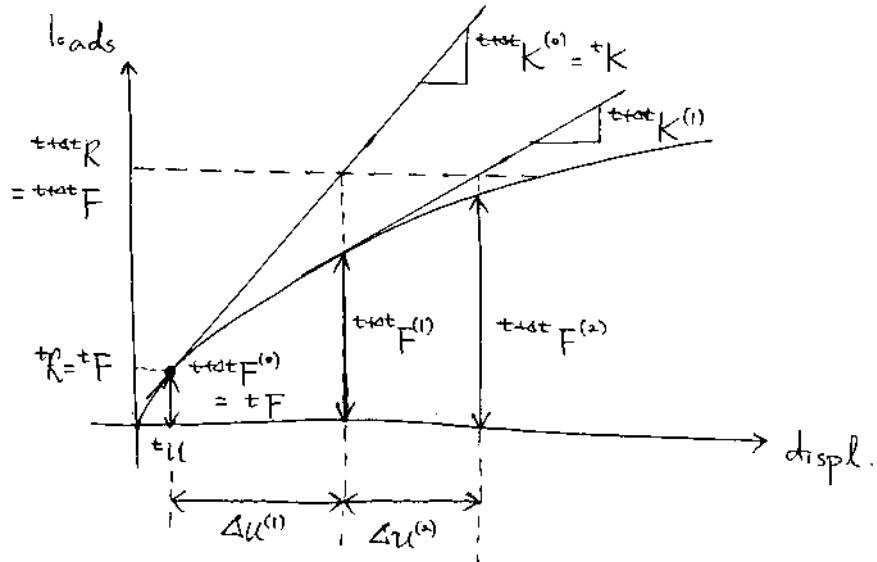
$${}^{t+\Delta t}\mathbf{K}^{(i-1)} = \frac{\partial {}^{t+\Delta t}\mathbf{F}}{\partial \mathbf{U}} \Big|_{{}^{t+\Delta t}\mathbf{U}^{(i-1)}} = \left(\frac{\partial \mathbf{F}}{\partial \mathbf{U}}\right) \Big|_{{}^{t+\Delta t}\mathbf{U}^{(i-1)}} \quad (18.21)$$

Physically



$${}^{t+\Delta t}K_{11}^{(i-1)} = \frac{\Delta \left({}^{t+\Delta t}F_1^{(i-1)}\right)}{\Delta u} \quad (18.22)$$

Pictorially for a single degree of freedom system



$$i = 1; \quad {}^t K \Delta u^{(1)} = {}^{t+\Delta t} R - {}^t F \quad (18.23)$$

$$i = 2; \quad {}^{t+\Delta t} K^{(1)} \Delta u^{(2)} = {}^{t+\Delta t} R - {}^{t+\Delta t} F^{(1)} \quad (18.24)$$

Convergence Use

$$\|\Delta U^{(i)}\|_2 < \epsilon \quad (18.25)$$

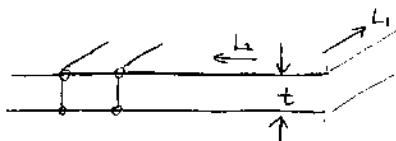
$$\|a\|_2 = \sqrt{\sum_i (a_i)^2} \quad (18.26)$$

But, if incremental displacements are small in every iteration, need to also use

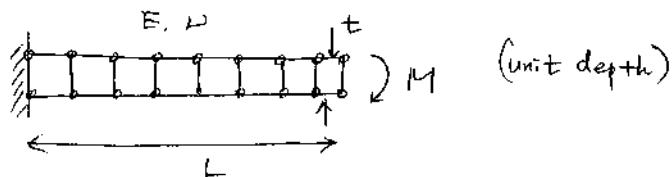
$$\|{}^{t+\Delta t} R - {}^{t+\Delta t} F^{(i-1)}\|_2 < \epsilon_R \quad (18.27)$$

18.1 Slender structures

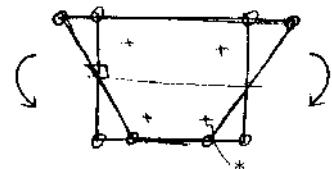
(beams, plates, shells)



$$\frac{t}{L_i} \ll 1 \quad (18.28)$$

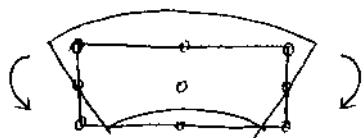
Beam

$$\text{e.g. } \frac{t}{L} = \frac{1}{100}$$



(4-node el.)

The element does not have curvature → we have a spurious shear strain



(9-node el.)

→ We do not have a shear (better)

→ But, still for thin structures, it has problems like ill-conditioning.

⇒ We need to use beam elements. For curved structures also spurious membrane strain can be present.

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2.094 Finite Element Analysis of Solids and Fluids II

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